

CLOSED-FORM SOLUTION EXAMPLE:

CHARACTERISTIC EQ'N:

$$([k] - \omega^2 [m]) \{q\} = \{0\}$$

APPLY SINGLE "BEAM" ELEMENT STIFFNESS & MASS MATRIX:

$$\left(EI \begin{bmatrix} 12 & -6L \\ -6L & 4L^2 \end{bmatrix} - \omega^2 \left(\frac{PAL}{420} \right) \begin{bmatrix} 156 & -22L \\ -22L & 4L^2 \end{bmatrix} \right) \begin{Bmatrix} v_2 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$\begin{Bmatrix} v_2 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

FOR NON-TRIVIAL SOLUTION, DETERMINANT OF EXPRESSION INSIDE

$$([] - \omega^2 []) = 0$$

THUS:

$$\begin{vmatrix} 12 - 156a & -6L + 22La \\ -6L + 22La & 4L^2 - 4La^2 \end{vmatrix} = 0$$

SOLVE TO OBTAIN:

$$\omega_1 = 3.533 \left(\frac{EI}{PAL^4} \right)^{1/2}$$

$$\omega_2 = 34.81 \left(\frac{EI}{PAL^4} \right)^{1/2}$$

$$a = \frac{\omega^2 PAL^4}{420EI}$$

MODE 1 SET $v_2 = 1$

SOLVE FOR θ_2

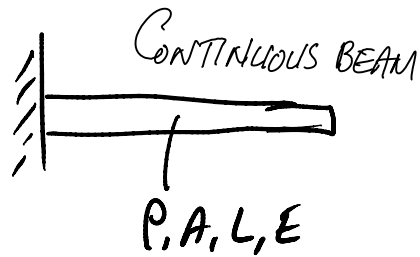
$$\theta_2 = \frac{1.38}{L}$$



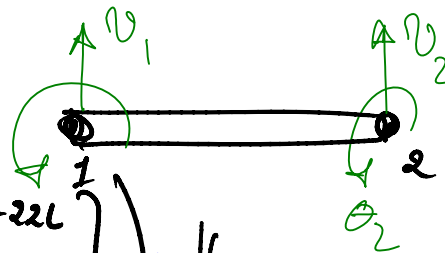
MODE 2

SET $v_2 = 1$

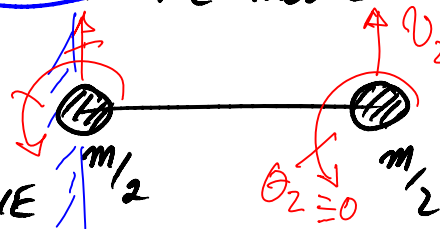
$$\theta_2 = \frac{2.62}{L}$$



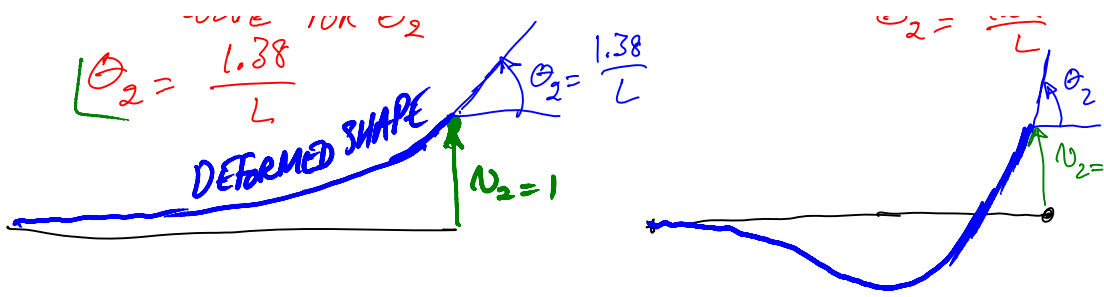
ONE ELEM. APPROX.



F.E. MODEL



NO ROTATIONAL INERTIA



LUMPED MASS APPROXIMATION:

$$\omega_1 = 2.449 \left(\frac{EI}{PAL^4} \right)^{1/2} \quad \omega_2 = 0$$

ANALYTICAL (CLOSED FORM)
SOLUTION

$$\omega_1 = 3.516 \left(\frac{EI}{PAL^4} \right)^{1/2}$$

$$\omega_2 = 22.03 \left(\frac{EI}{PAL^4} \right)^{1/2}$$

